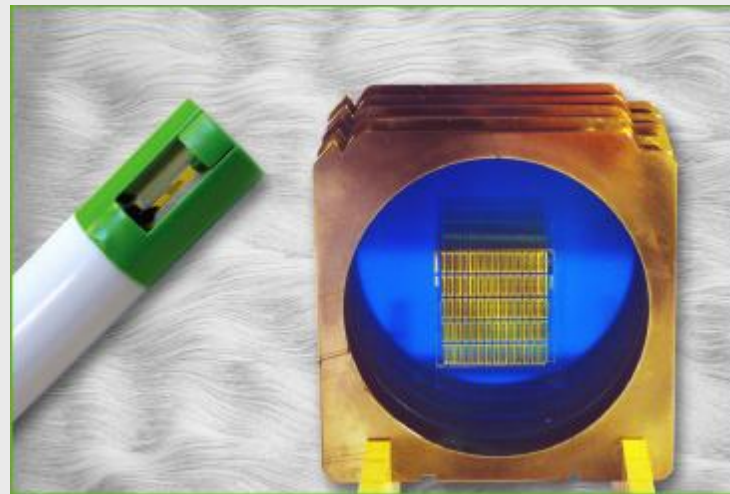


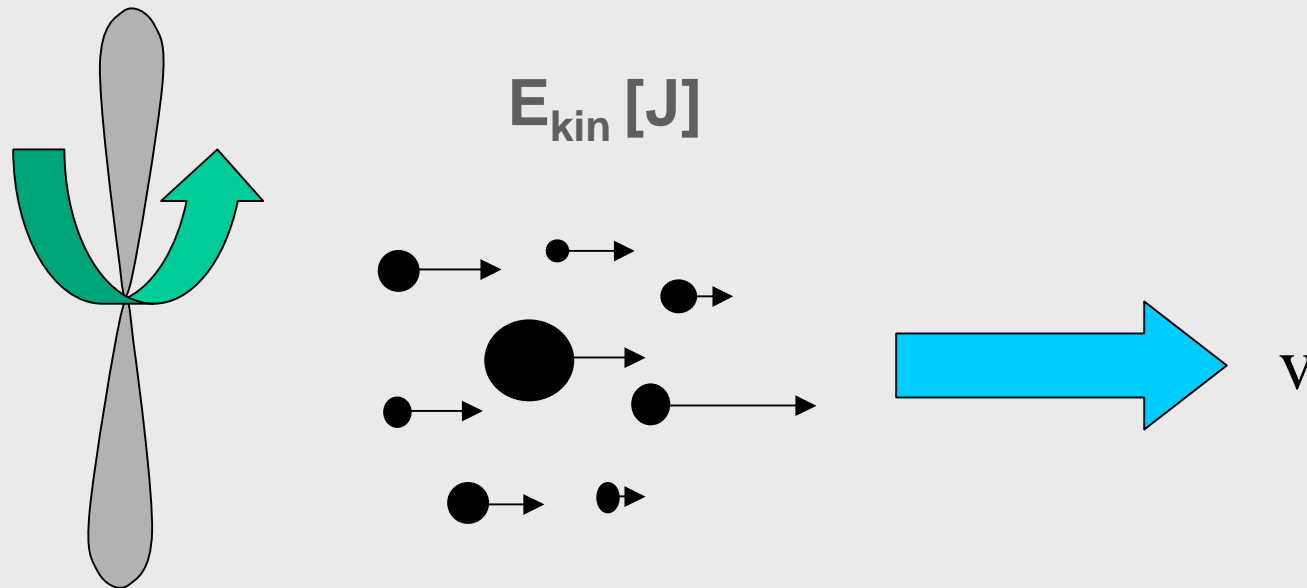
AIR FLOW MEASUREMENT

- ERIK THOR, Ph.D.
- BASIC DEVELOPMENT / FLUID MECHANICS



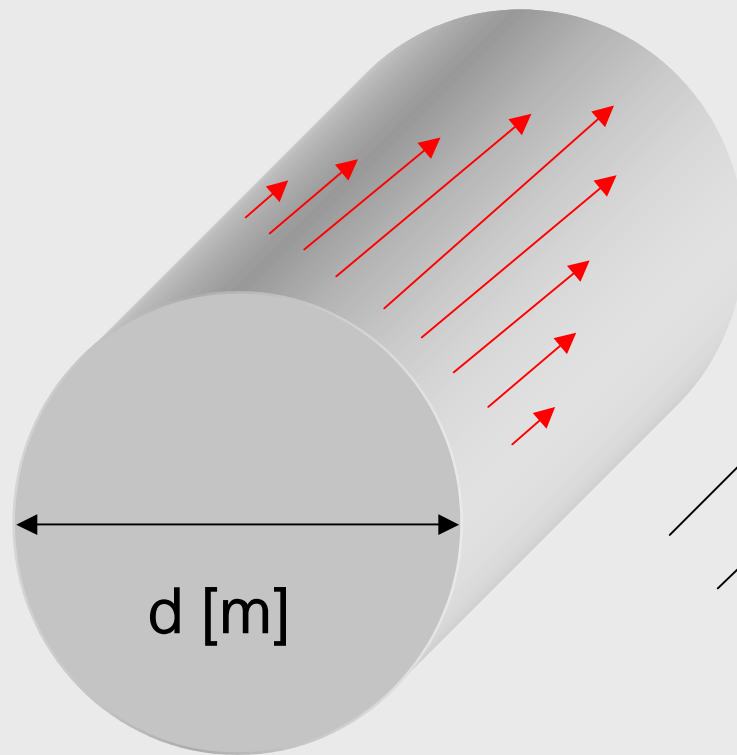
AIR FLOW PHYSICS

Air = Mixture of gas molecules (N_2 , O_2 , Ar, CO_2 , ...)

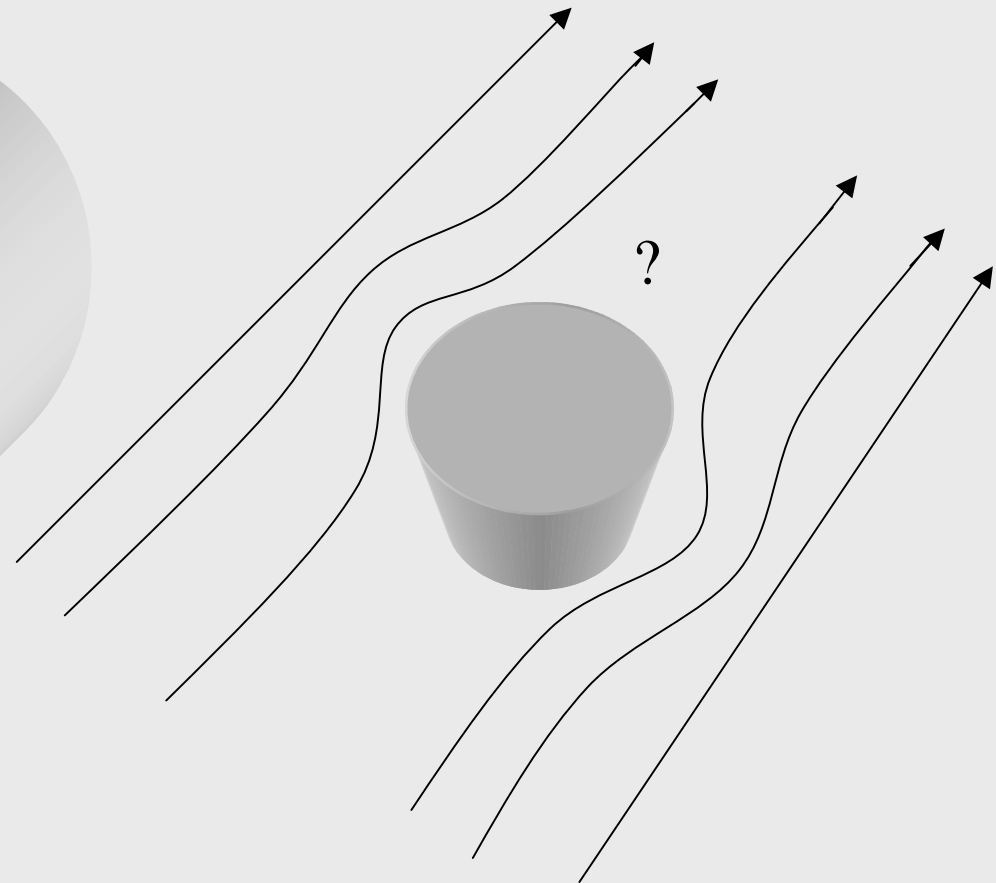


Air velocity v [m/s] =
mean travelled distance per time of an air molecule

closed channel air flow



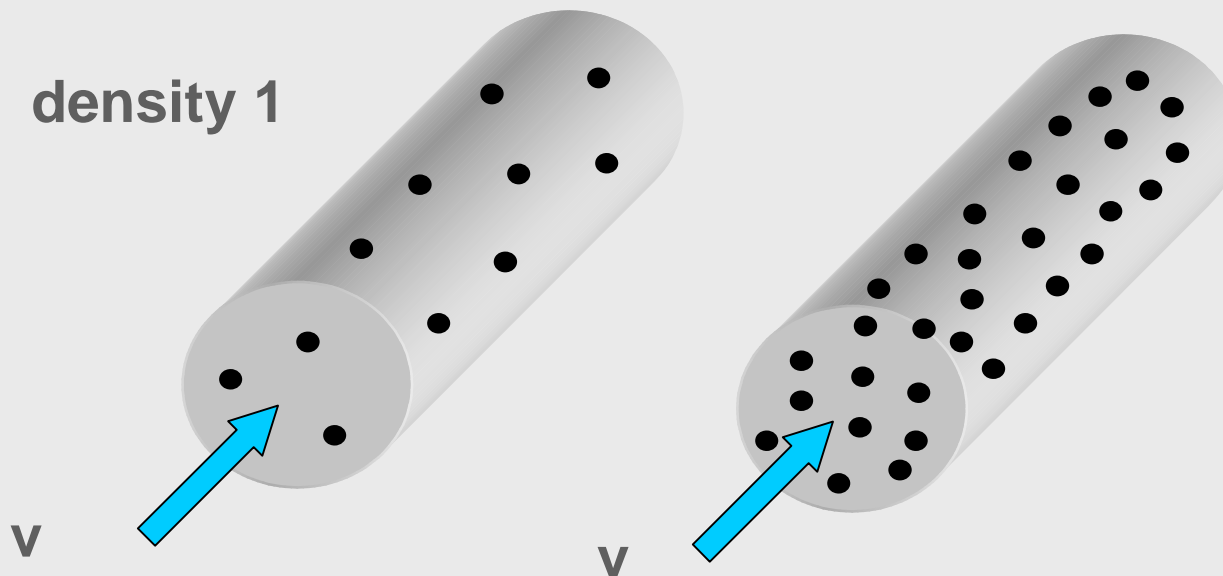
air flow past immersed body



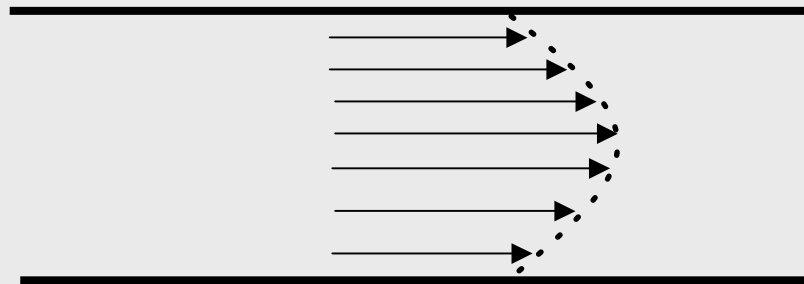
Mass flow = transferred mass per time and area [kg/(m²×s)]

= Air Density [kg/m³] × Air Velocity [m/s]

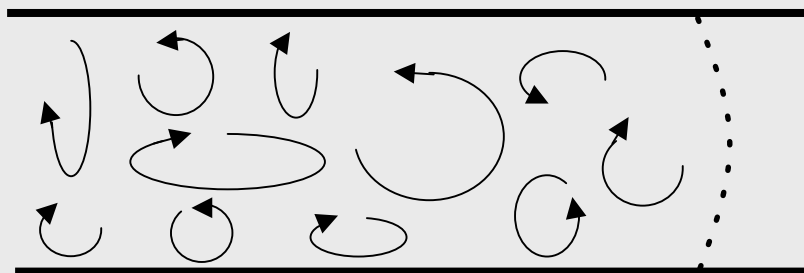
density 2 > density 1



„low“ velocity ® regular (“ laminar “) flow



„high“ velocity ® irregular (“ turbulent flow “)



“low - high“ velocity ® „Reynolds number“

Re= inertial force / viscous force

$$\text{Re} = \frac{d \cdot v \cdot \rho}{\eta} = \frac{\text{diameter} [m] \cdot \text{velocity} \left[\frac{m}{s} \right] \cdot \text{density} \left[\frac{kg}{m^3} \right]}{\text{viscosity} \left[\frac{kg}{m \cdot s} \right]}$$

Re < 2000 for laminar flow in a pipe

TYOLOGY OF (AIR) FLOWMETERS



Flowmeter Type	Liquid	Gas	Steam	Pipe size	Clean Fluid	Dirty Fluid
Coriolis	x	x		1/6 to 6 in,	x	Ltd.
Differential Pressure	x	x	x	½ in. and up	x	Ltd.
Magnetic	x			1/10 to 100 in.	x	x
Positive Displacement	x	x		2 – 30 in.	x	
Thermal (HW, HF, Calorimetric)	Ltd.	x		Insertion	x	no problem for HF !
Turbine	x	x		2 – 30 in.	x	
Ultrasonic-Transit Time	x	x	Ltd.	½ inch and up	x	Ltd.
Ultrasonic-Doppler	x			½ inch and up		x
Vortex	x	x	x	½ inch to 12 in.	x	

Flowmeter Type	Advantages	Disadvantages
Coriolis	High accuracy; Low maintenance; Insensitive to flow profile	High initial cost, depending on size and model; Bent tubes subject to fouling; Not available for pipe sizes over six inches
Differential Pressure	Low initial cost; Ease of installation; Well understood; Many industry approvals	Limited rangeability; Permanent pressure drop; Uses square root method to calculate flowrate; Requires periodic maintenance
Magnetic	Obstructionless; High accuracy; No pressure drop	Cannot meter nonconductive fluids (e.g., hydrocarbons); Relatively high initial cost; Electrodes subject to coating
Positive Displacement	High accuracy; Insensitive to flow profile; High rangeability	Cannot handle dirty fluids; Subject to wear; Pressure drop
Thermal (Calorimetric, Hot Wire, Hot Film)	Relatively low initial cost; Good for low velocity flows	Limited accuracy; Sensitive to problems of dirty fluids except HF
Turbine	High accuracy; Well-known technology; Medium purchase price	Cannot handle dirty fluids; Bearings subject to wear; Pressure drop
Ultrasonic-Transit Time	High accuracy, depending on model; Obstructionless; Clamp-on convenience; No pressure drop	Limited ability to handle dirty fluids; Can be affected by flow profile; Some models have high initial cost
Ultrasonic-Doppler	Can meter dirty flows; No pressure drop; Clamp-on convenience	Low to medium accuracy; Reynolds number limitations
Vortex	Highly versatile: can measure liquid, gas, and steam; Good accuracy	Limited ability to handle low flows; Vibration can affect accuracy; Few industry approvals



Rate the importance of factors you use when specifying or purchasing flowmeters (scale is 1 to 5)

Criteria	Ranking	Minimum	Maximum
Reliability	4.81	4	5
Compatibility	4.80	4	5
Repeatability	4.78	4	5
Application	4.69	2	5
Technical support	4.67	3	5
User friendly/simplicity	4.60	4	5
Accuracy	4.45	1	5
Maintainability/Repair	4.30	3	5
Price	3.77	1	5

Jesse Joder, "Flowmeter Shootout Part III", <http://www.flowresearch.com/articles.htm>

HOT FILM TECHNOLOGY

EE65

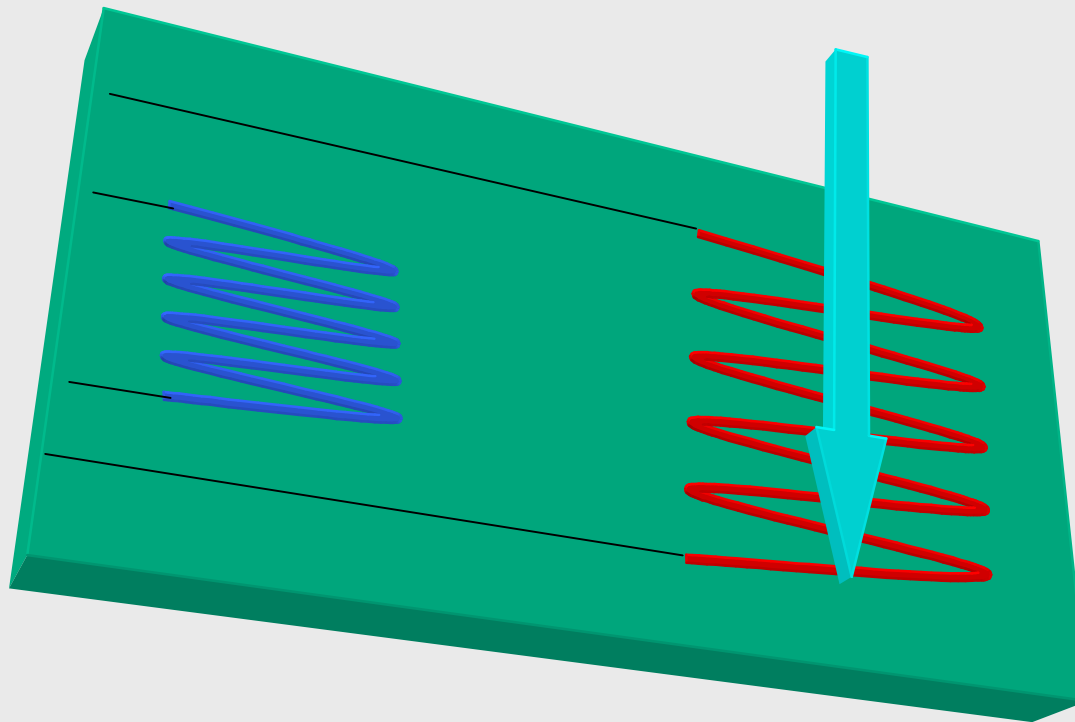


EE70



Temperature Sensor

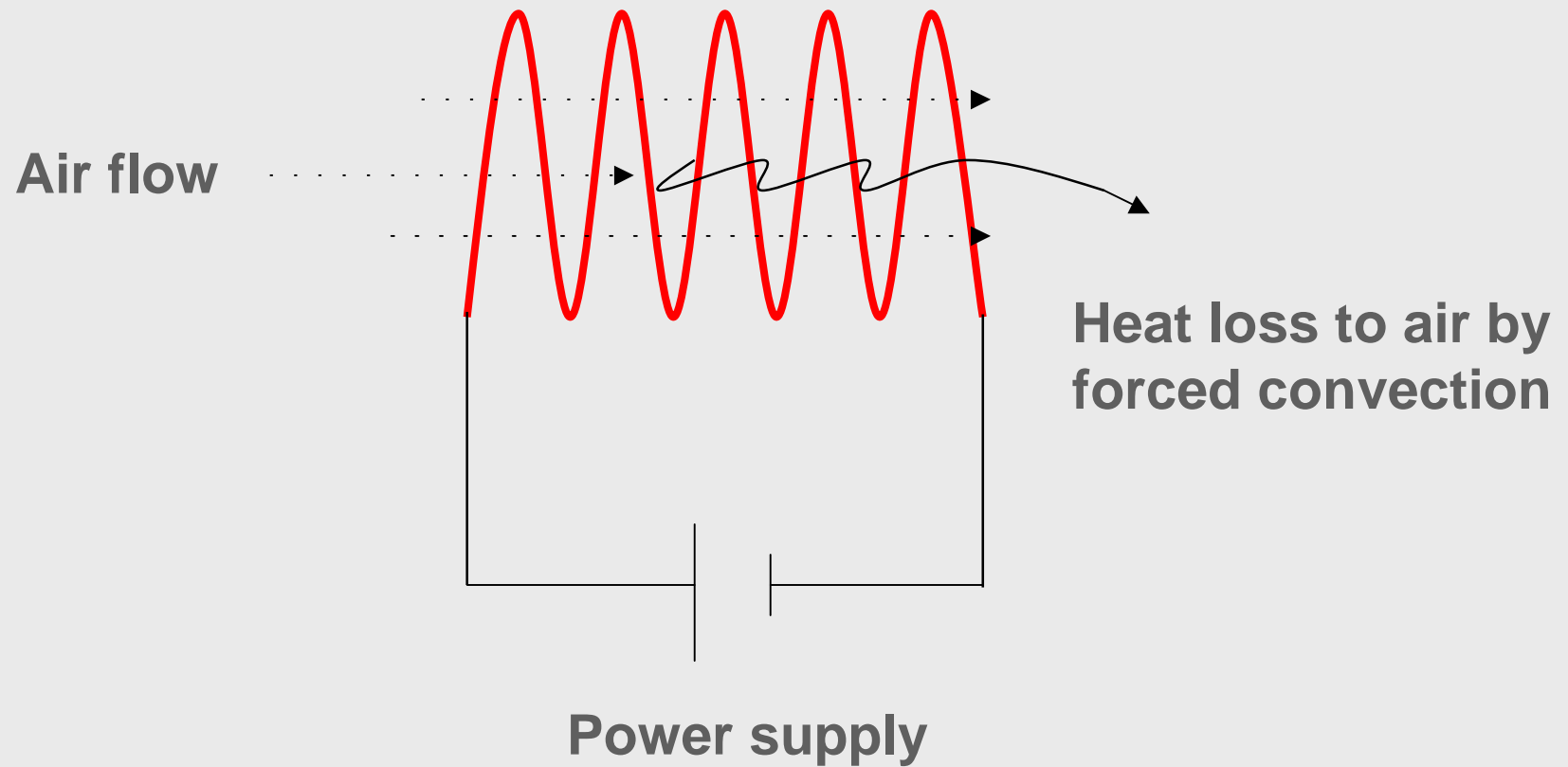
Velocity Sensor



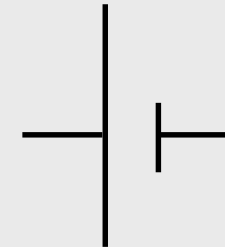
Structured Mo metal film (@ 1 μm) on thin glass substrate (150 μm) coated by polyimide protective layer

HOT FILM MEASUREMENT PRINCIPLE

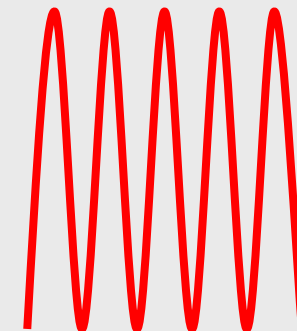
Hot meander film



- Power supply $\text{P} = U \times I$



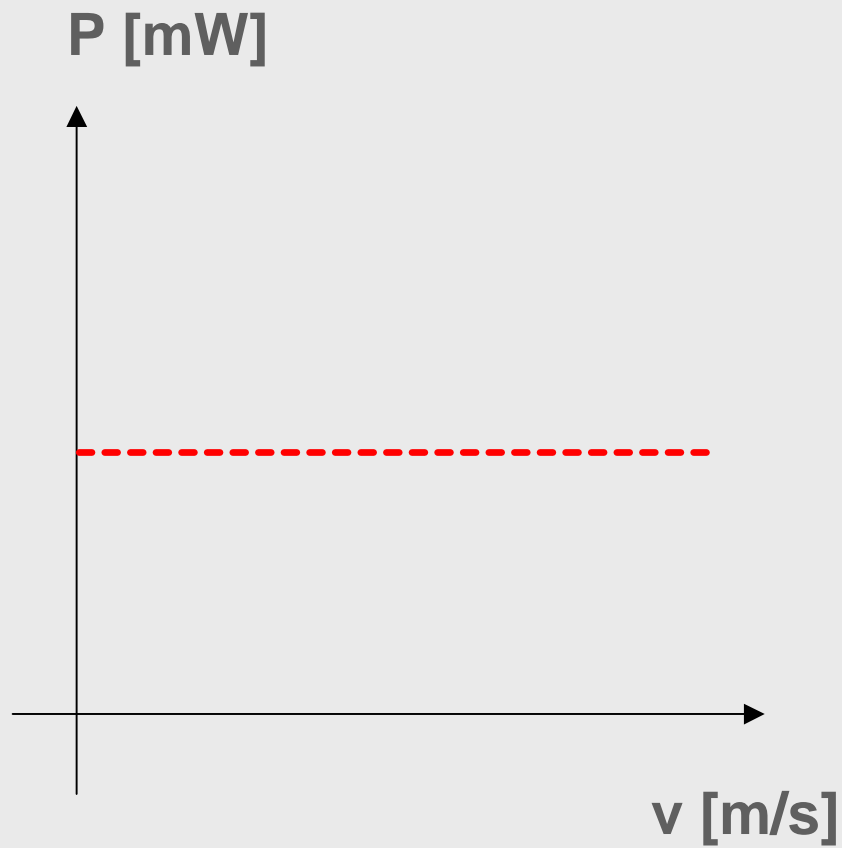
- Hot film resistance R_h
heater temperature T_h



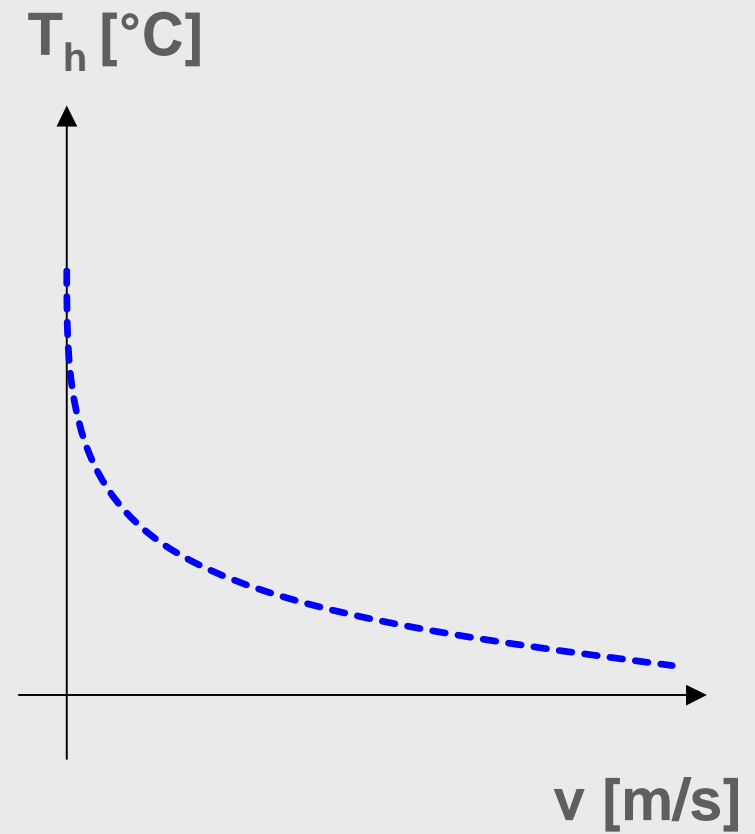
- Air flow
velocity v , ambient temperature T_a



Power constant

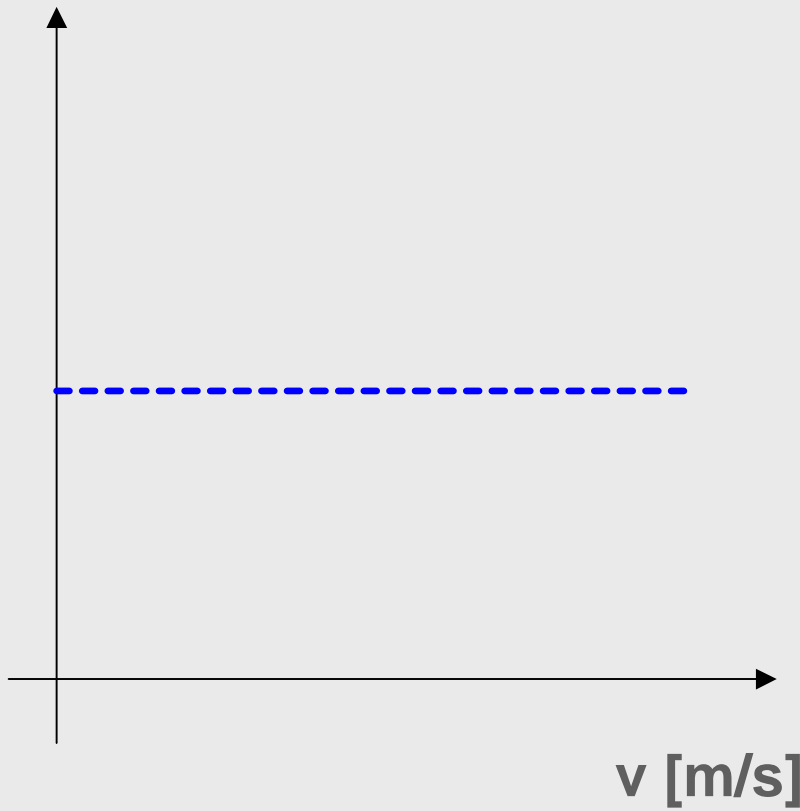


Heater cools down



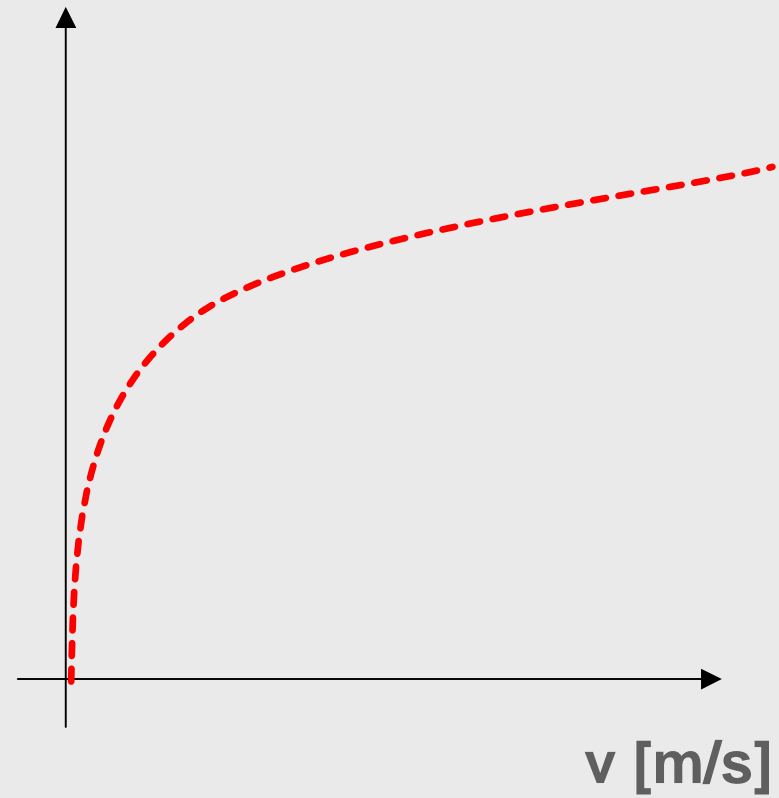
Heater temperature constant

T_h [°C]

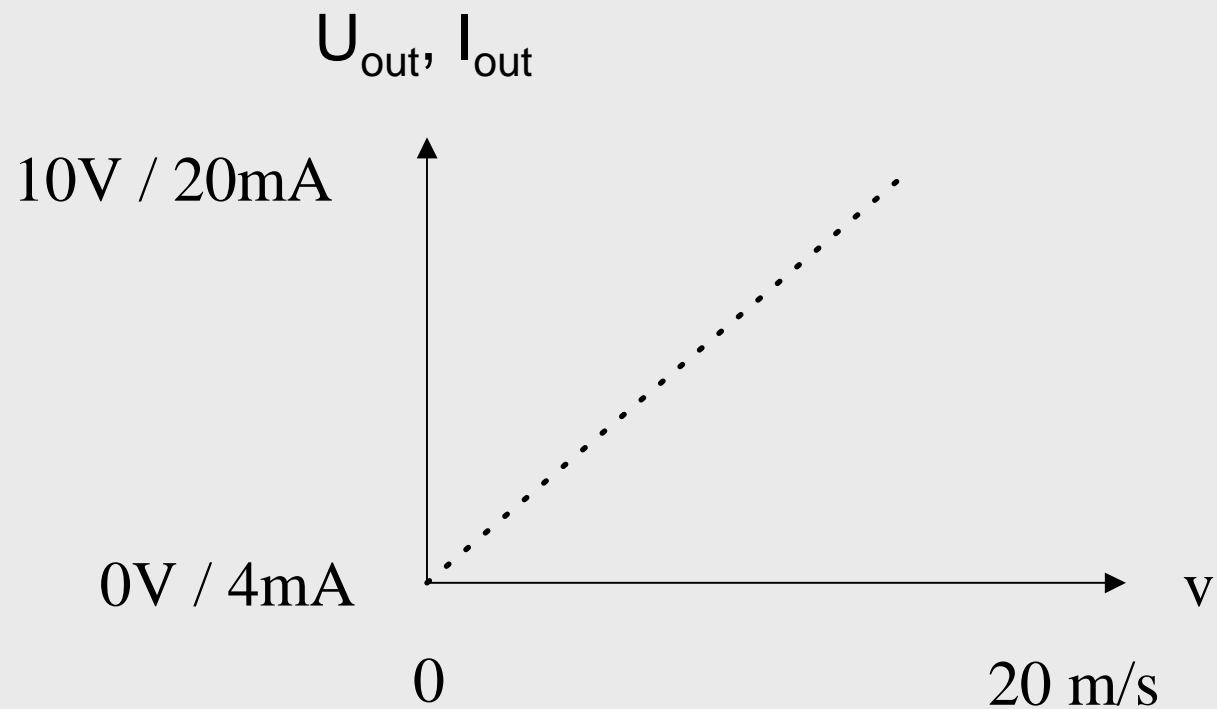


Power function of velocity

P [mW]



Electronic unit converts $P(v)$ into
linearized signal $U_{out}(v)$, $I_{out}(v)$
for a defined velocity region



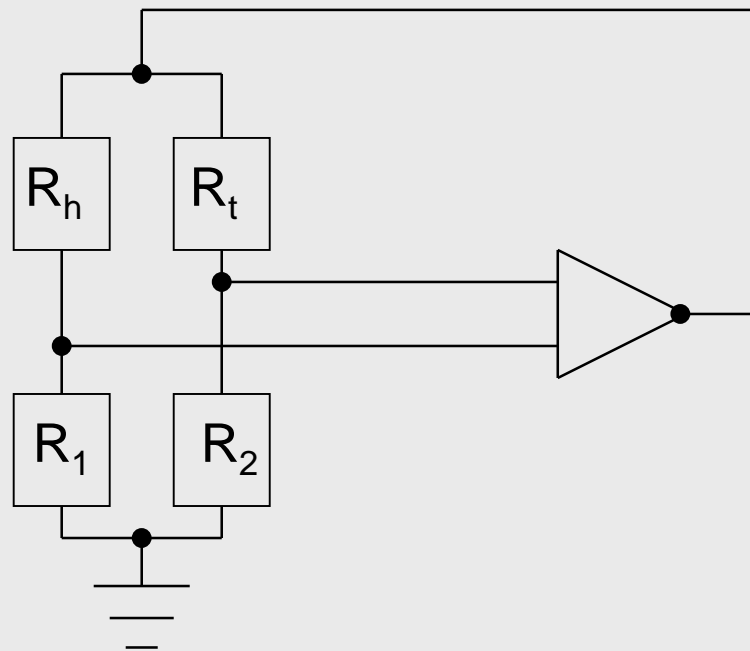
CLIMATIC INFLUENCES

Convective heat transmission depends on temperature difference between hot film and ambient air !

$$\frac{dQ}{dt} [W] \propto T_h - T_a$$

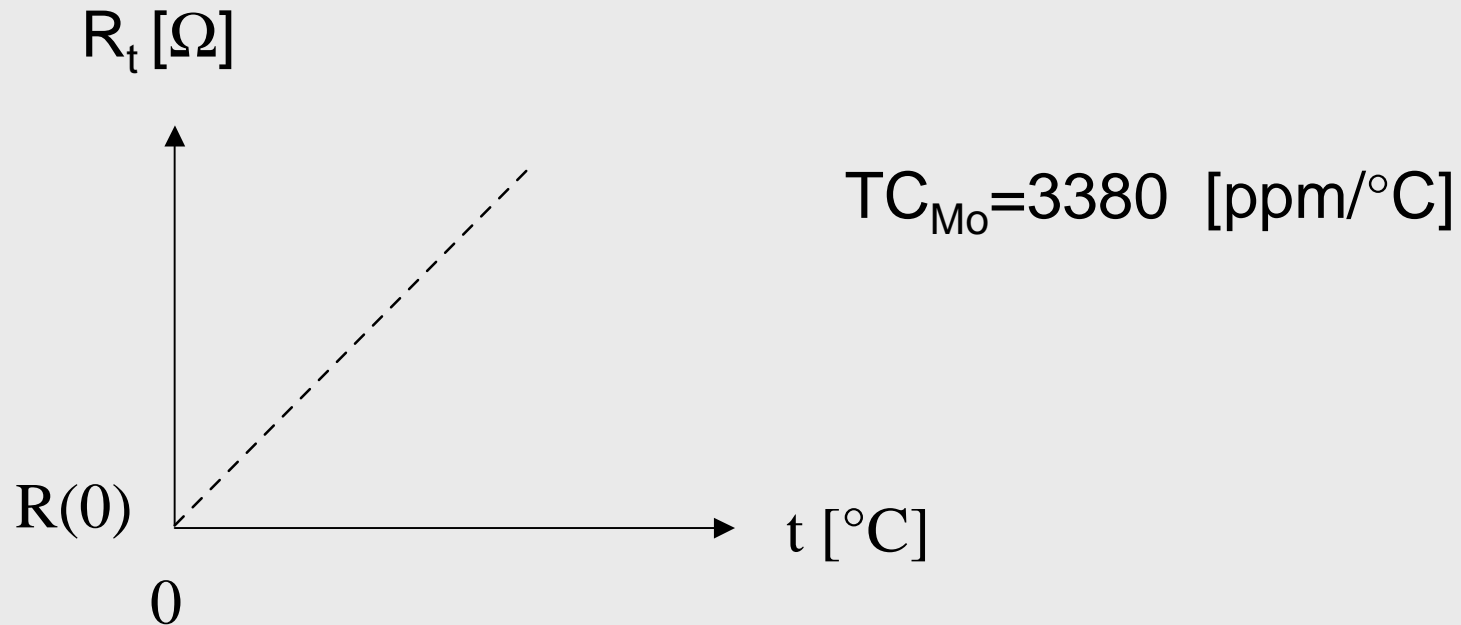
$T_h - T_a$ kept constant by

- Microcontroller (EE70)
- Wheatstone bridge circuit (EE65)

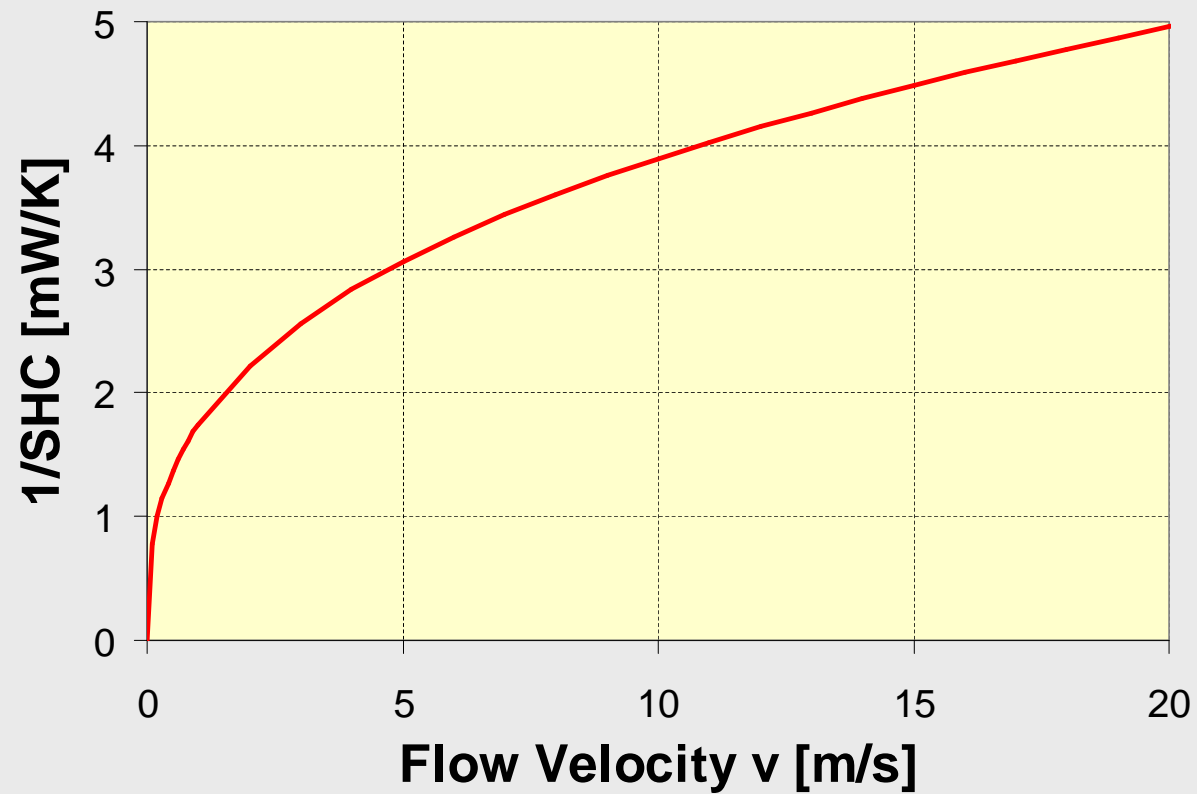


Ⓜ Temperature Measurement

$$R_t(t) \cong R(0) \cdot (1 + TC \cdot t)$$



$$\frac{1}{SHC} \equiv \frac{P(v)}{T_h - T_a} = B \cdot v^n$$



**1/SHC-characteristic depends on
air fluid properties i.e.**

- **air density ρ [kg/m³] !!!**
- **thermal conductivity λ [W/(m×K)]**
- **viscosity η [kg/(m×s)]**
- **thermal capacity c_p [J/(kg×K)]**

***AIR PRESSURE -
AN UNCOMPENSATED INFLUENCE***

Continuity equation + Ideal gas equation

$$\frac{1}{A} \cdot \frac{dm}{dt} [\text{kg} / (\text{m}^2 \cdot \text{s})] = r \cdot v$$

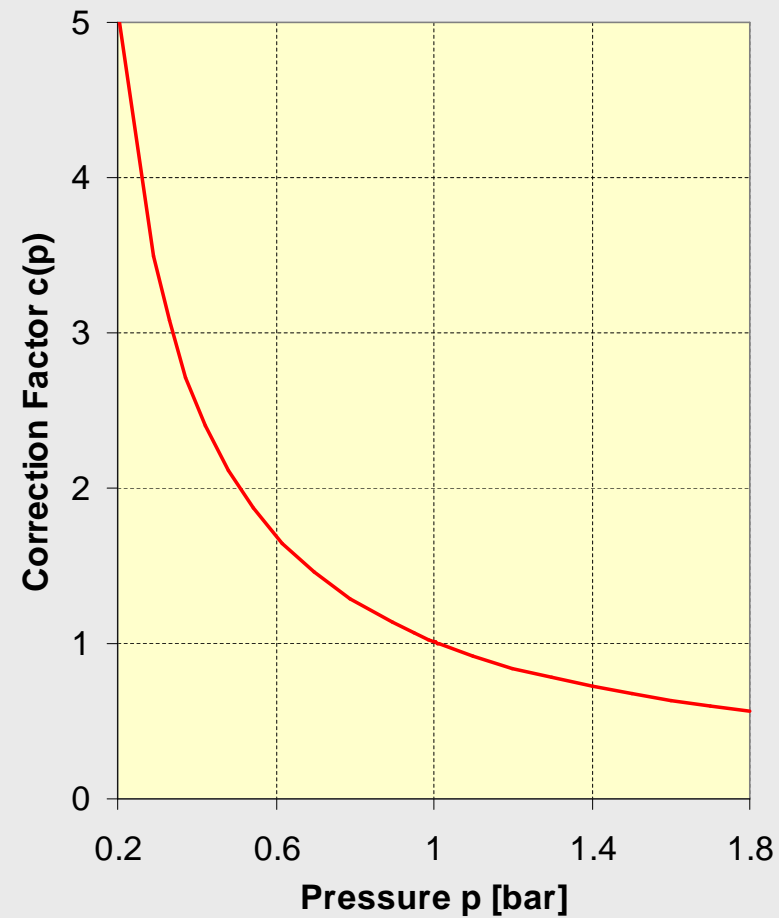
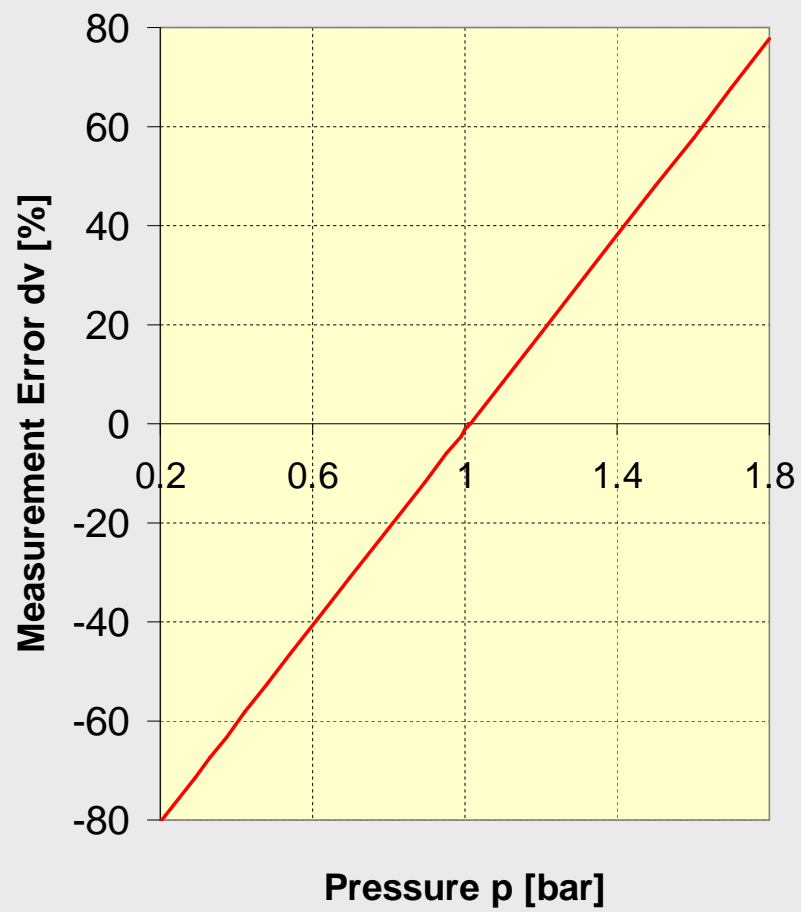
$$r [\text{kg} / \text{m}^3] = r_0 \cdot \frac{p}{T} \cdot \frac{T_0}{p_0}$$

Measured velocity v_m

„True“ velocity v

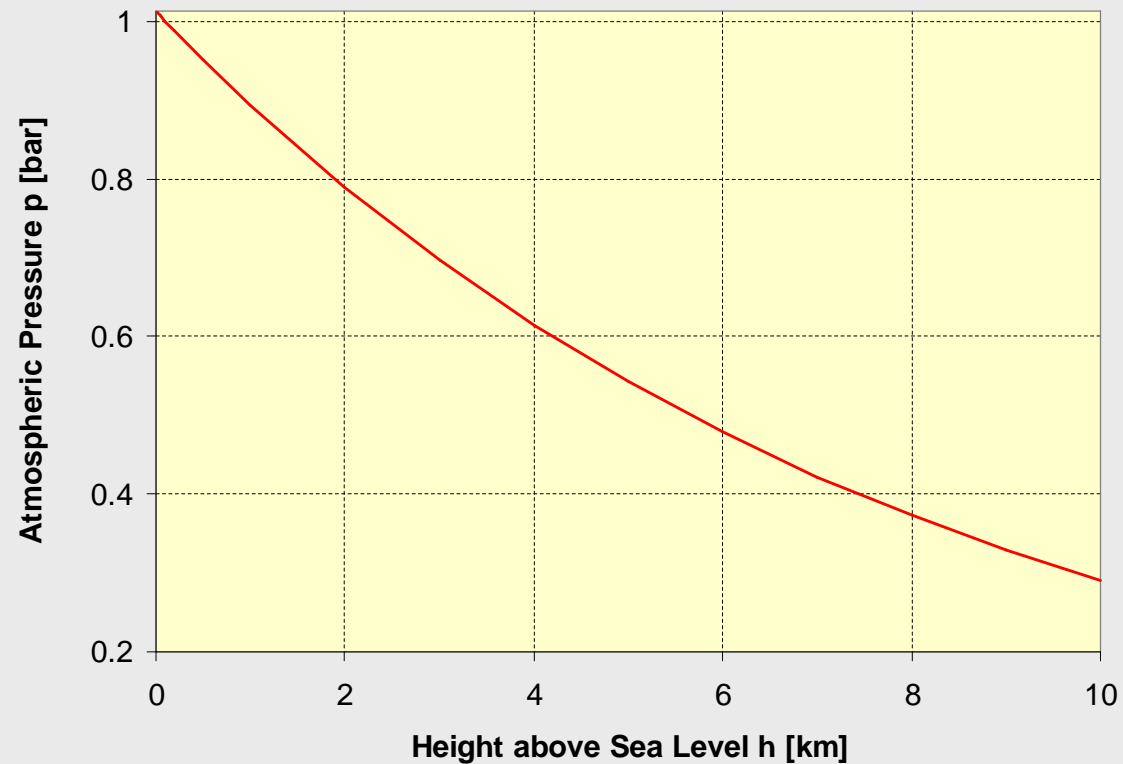
$$v_m = v \cdot \frac{p}{p_0} \rightarrow$$

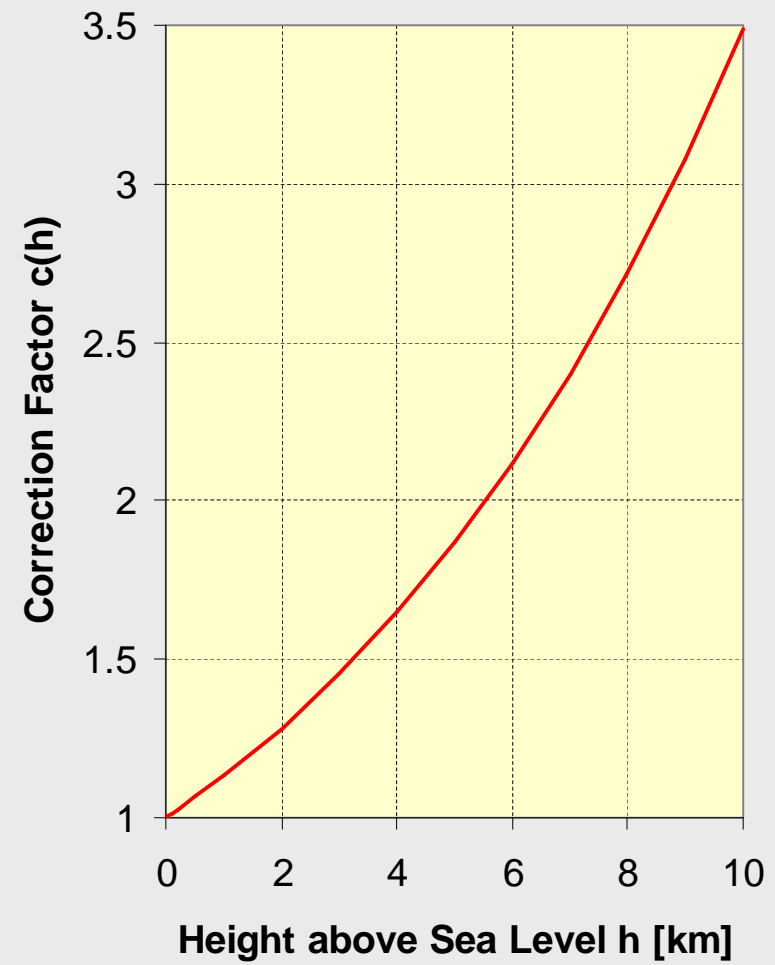
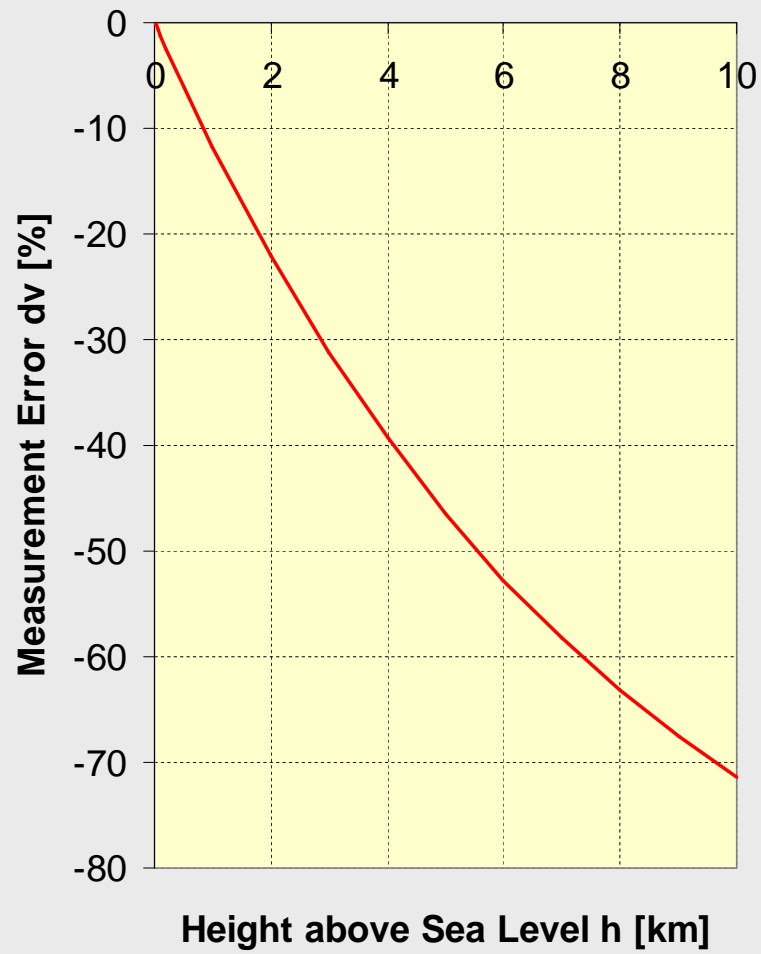
$$v = v_m \cdot \frac{p_0}{p} = v_m \cdot c(p)$$



Atmospheric air pressure p depends on
height h above sea level

$$p = p_0 \cdot e^{-\frac{h}{8km}}$$





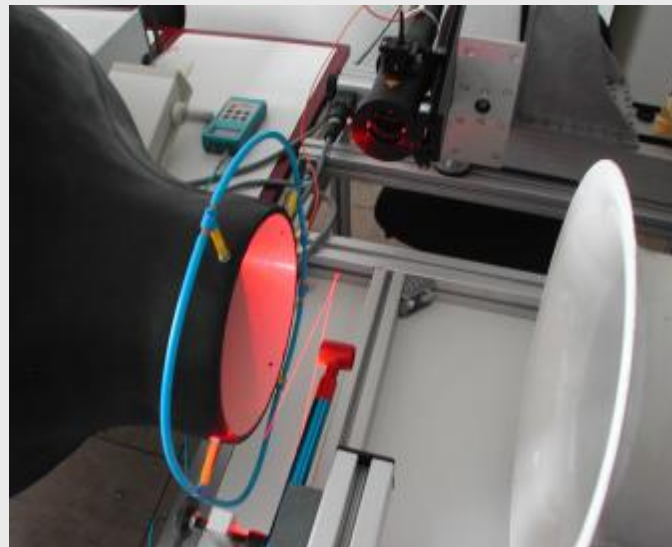
CALIBRATION - WIND TUNNEL TESTING

CALIBRATION

1. Velocity measurement device under test (VDUT)

2. Certificated velocity reference

calibration = measurement of the deviations of the VDUT
from the reference under controlled flow conditions



E+E primary velocity reference ®

Laser-Doppler Anemometer (LDA) certificated by PTB



with only 0.2 % uncertainty in velocity measurement

- Wind Tunnel for controlled flow conditions



Open Jet



Closed Jet

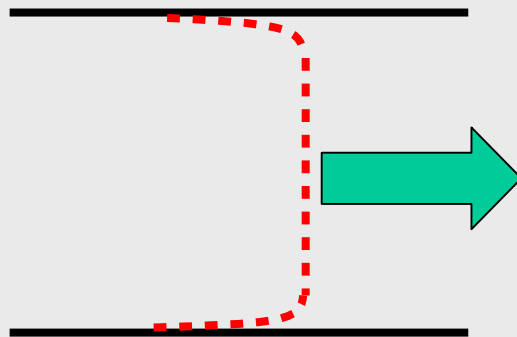
$$v=1 \text{ m/s}$$

$$d=0.2 \text{ m}$$

$$\rho_{\text{Air}} (20^\circ\text{C}, 1\text{bar}) = 1.188 \text{ kg/m}^3$$

$$\eta_{\text{Air}} (20^\circ\text{C}, 1\text{bar}) = 1.824 \times 10^{-5} \text{ kg/(m}\times\text{s)}$$

Ⓜ Re=13026 >> 2000 Ⓜ turbulent flow



special tunnel design for
laminar, uniform flow in
test section

At E+E two closed jets with

(a) closed test section (“Göttingen type“)

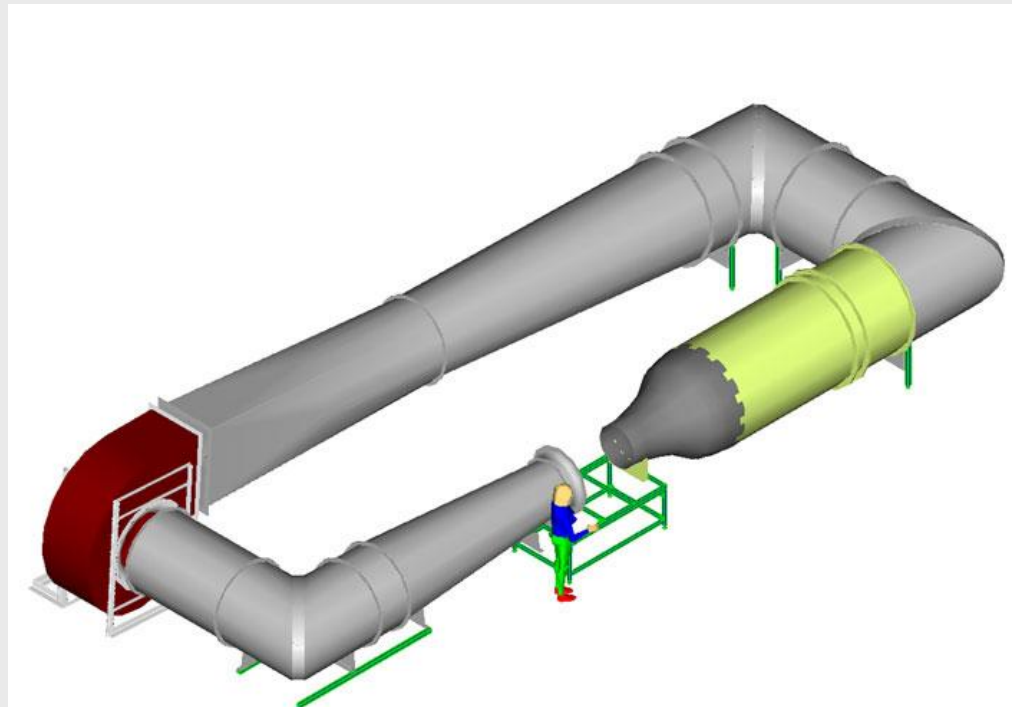
$v=0-30 \text{ m/s}$

$T_a=10-40^\circ\text{C}$

(b) open test section
 (“Prandtl type“)

$v=0-40 \text{ m/s}$

$T_a=RT=23^\circ\text{C}$



WIND TUNNEL FOR MEASUREMENT OF TRANSMITTER CHARACTERISTICS

Testing of transmitter performance under different

- climatic conditions ® temperature, pressure, humidity
- geometric conditions ® twisting angle, probe position in the flow
- dynamic conditions ® response time

SUDDEN CHANGE IN...

(a) Velocity at constant temperature (HVAC) ®

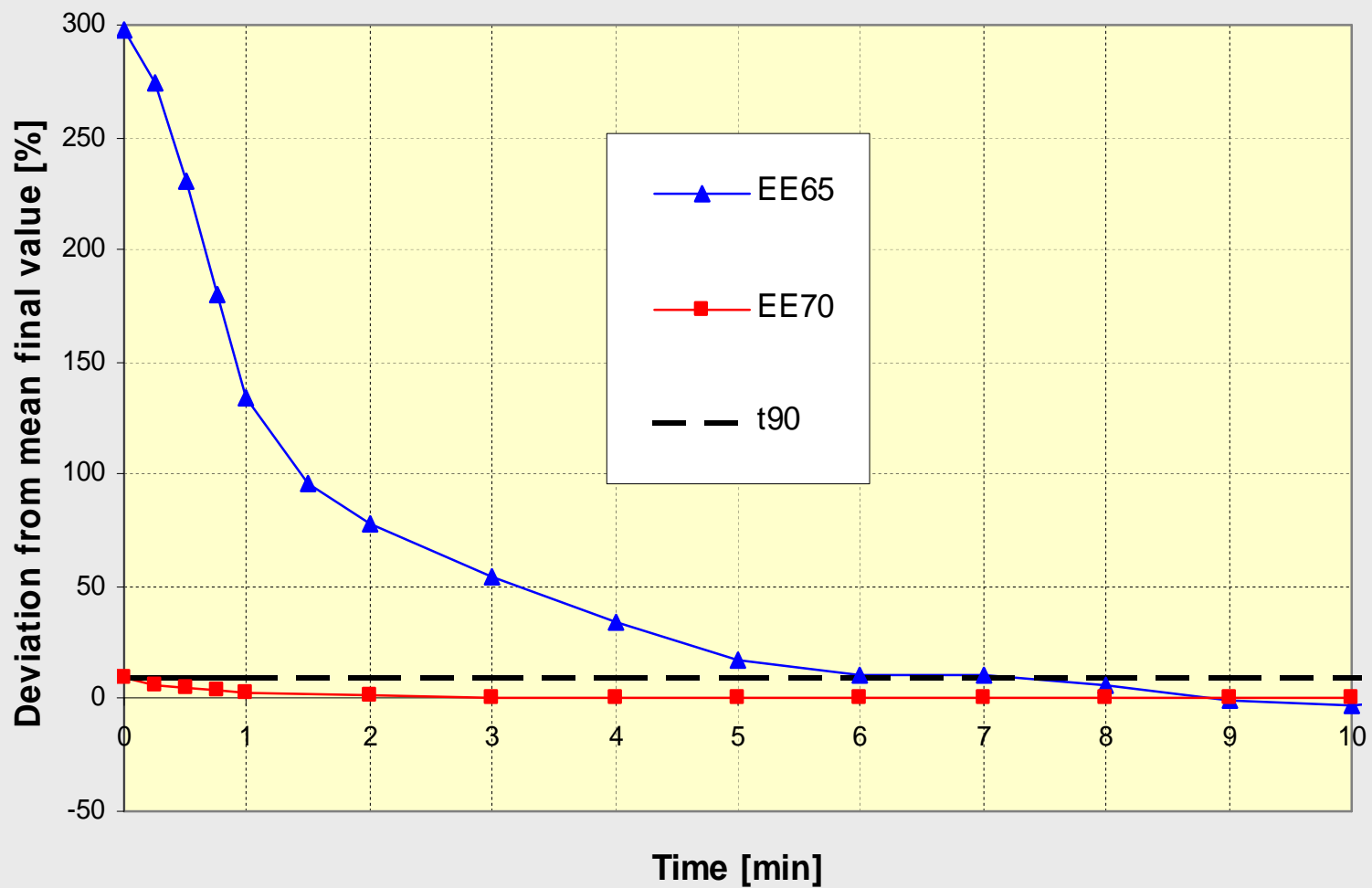
ü EE65 (0.2/2s), EE70 (1.5s)

(b) Velocity and temperature ®

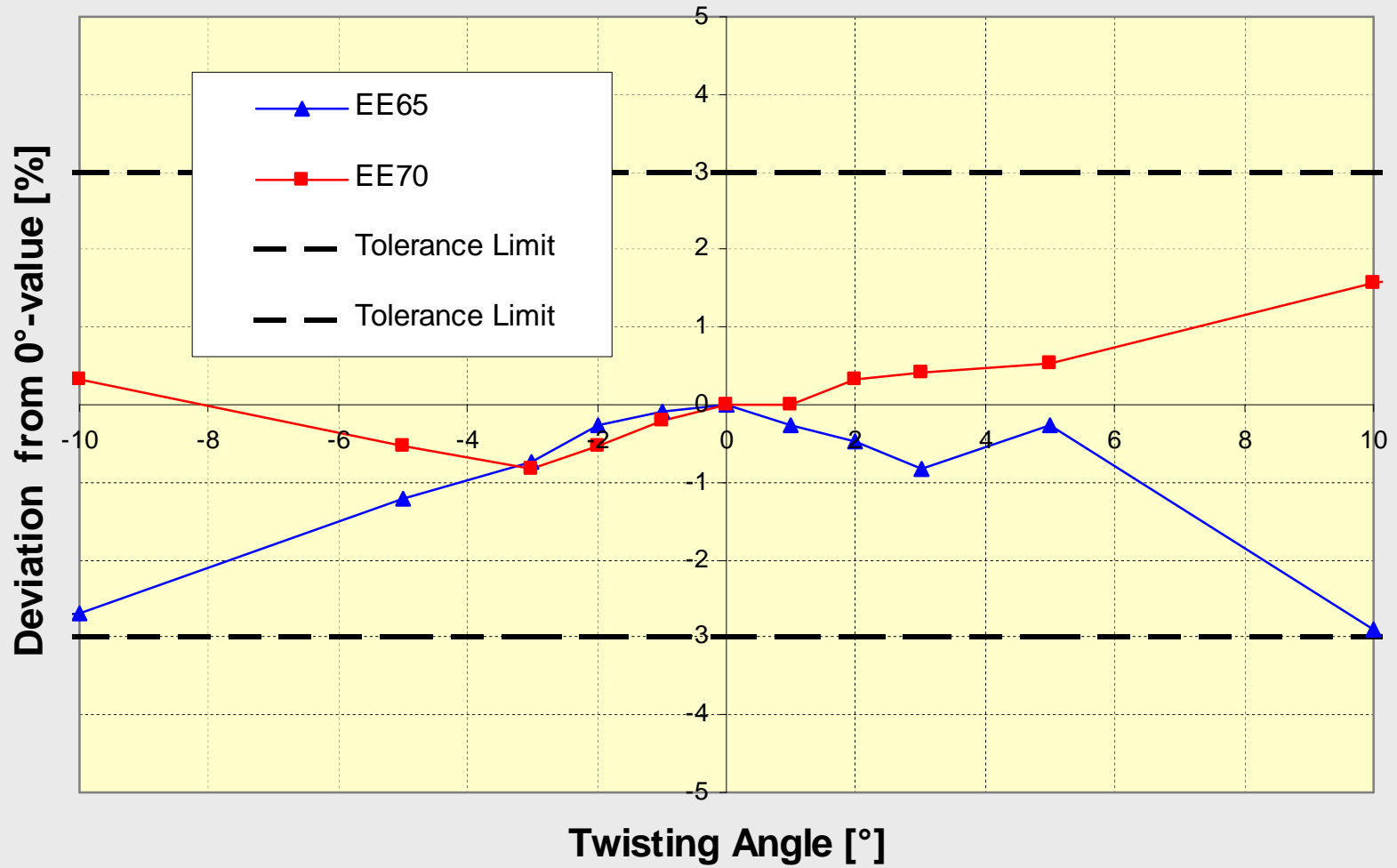
ü EE70

EE65 optional but slow

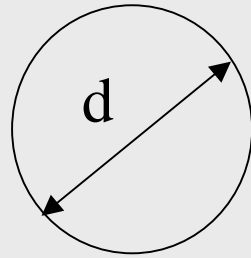
Response times for a change from (0m/s,50°C) to (0.5m/s,23°C)



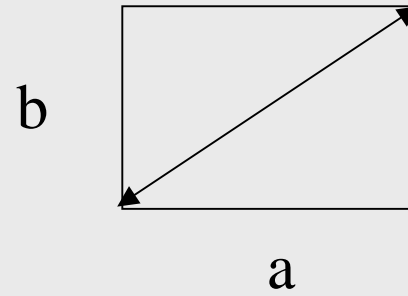
Direction Characteristics at 10 m/s



***GUIDELINES FOR GETTING
REALISTIC MEASUREMENT
RESULTS...***

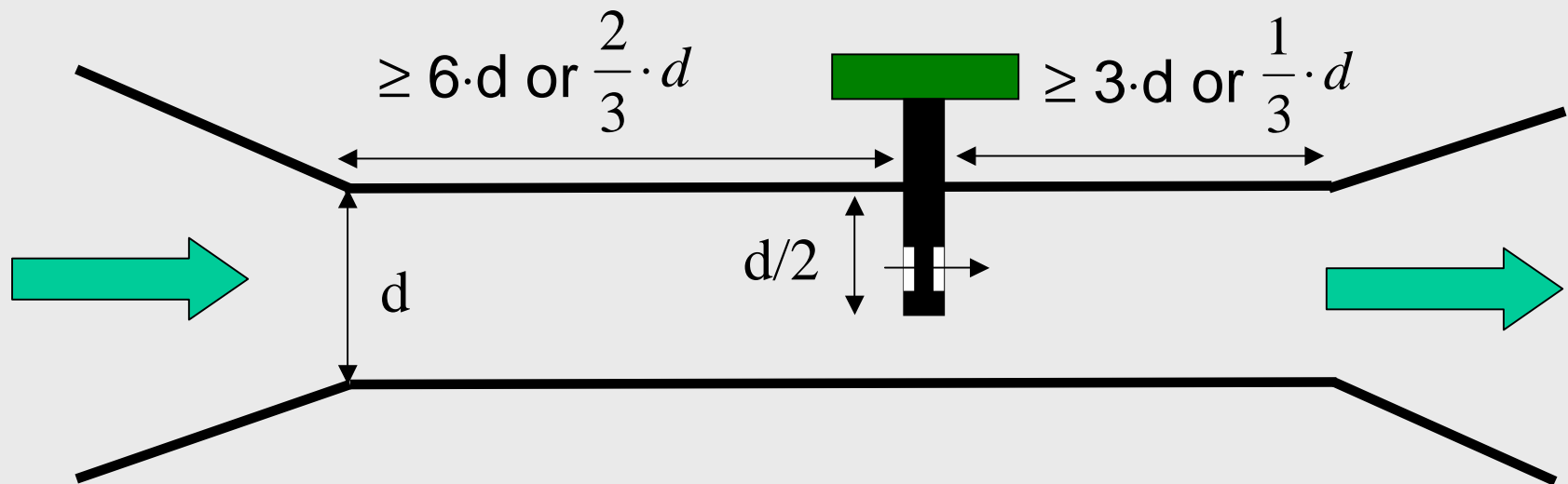


circular duct shape



$$d = \sqrt{a^2 + b^2}$$

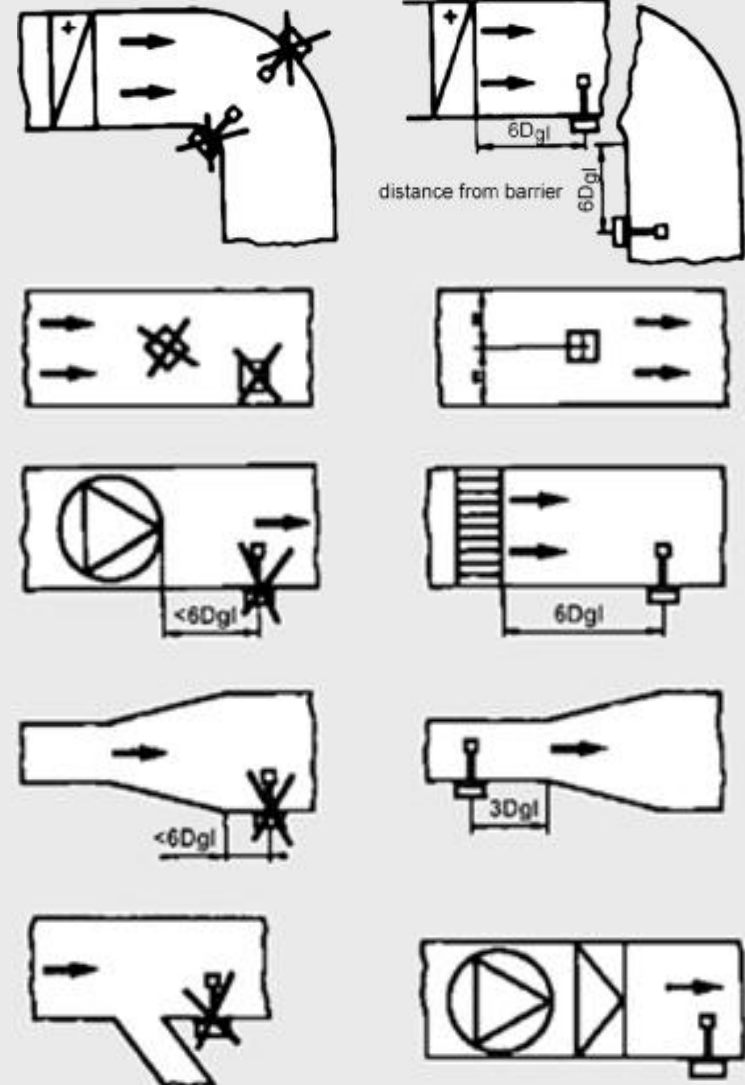
rectangular duct shape



Take care to...

- avoid corners
- avoid walls
- keep parallel to the flow
- avoid flow interrupting elements
- avoid contractions, diffusers

Dont's and Do's



*Thank you for
your attention !*

